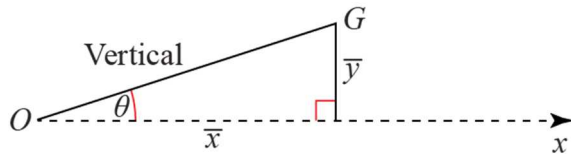


Centres of mass of plane figures 2F

1 a From question 1a in Exercise 2D,

$$\bar{x} = 2\frac{1}{2}; \bar{y} = \frac{13}{14}$$

Vertical



In equilibrium, G will be vertically below O i.e. OG is the vertical.

$$\tan \theta = \frac{\bar{y}}{\bar{x}} = \frac{\frac{13}{14}}{2\frac{1}{2}}$$

$$= \frac{13}{14} \times \frac{2}{5} = \frac{13}{35}$$

$$\theta = \tan^{-1}\left(\frac{13}{35}\right) = 20.4^\circ (3 \text{ s.f.})$$

b From question 1a in Exercise 2E,

$$\bar{x} = \frac{11}{4}; \bar{y} = \frac{5}{4}$$

$$\text{As above, } \tan \theta = \frac{\bar{y}}{\bar{x}} = \frac{\frac{5}{4}}{\frac{11}{4}}$$

$$\text{i.e. } \tan \theta = \frac{5}{11}$$

$$\theta = \tan^{-1}\left(\frac{5}{11}\right) = 24.4^\circ (3\text{s.f.})$$

c From question 1b in Exercise 2D.

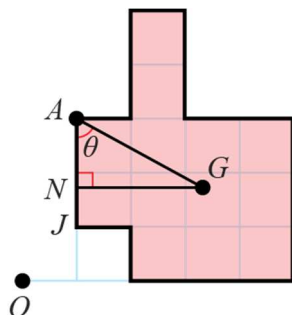
$$\bar{x} = 1.7; \bar{y} = 2.6$$

$$\tan \theta = \frac{2.6}{1.7} = \frac{26}{17}$$

$$\theta = \tan^{-1}\left(\frac{26}{17}\right) = 56.8^\circ (3 \text{ s.f.})$$

2 From question 4 in Exercise 2D,

$$\bar{x} = \frac{79}{26}; \bar{y} = \frac{51}{26}$$



These are the coordinates of the centre of mass, G , referred to O as origin.

A is the point of suspension.

$$G \text{ is } \left(\frac{79}{26}, \frac{51}{26} \right).$$

When the lamina hangs in equilibrium from A ,

AG will be the downward vertical.

Let N be the point on AJ such that GN is perpendicular to AJ .

See diagram.

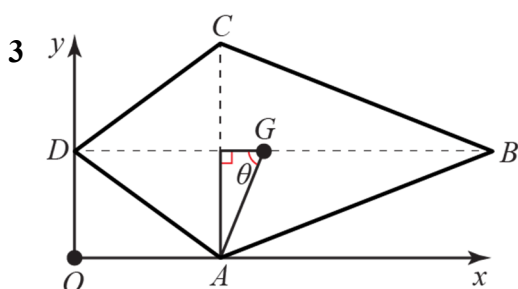
Then $\hat{NAG} = \theta$ is the required angle.

Since A is the point $(1, 3)$.

$$\begin{aligned} \tan \theta &= \frac{GN}{AN} = \frac{\bar{x} - 1}{3 - \bar{y}} \\ &= \frac{\frac{79}{26} - 1}{3 - \frac{51}{26}} = \frac{79 - 26}{78 - 51} \end{aligned}$$

Multiply top and bottom by 26.

$$= \frac{53}{27} \Rightarrow \theta = 63.0^\circ \text{ (3 s.f.)}$$



G , the centre of mass has coordinates $\left(\frac{7}{3}, 2 \right)$ taking O as origin.

Since AG will be vertical in equilibrium, the angle between AC and the horizontal will be θ .

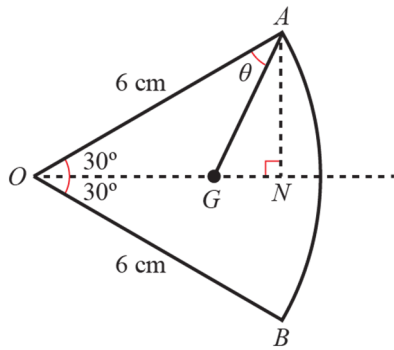
θ is the required angle

$$\begin{aligned} \tan \theta &= \frac{2}{\frac{7}{3} - 2} \\ &= \frac{6}{7 - 6} \\ &= 6 \end{aligned}$$

Multiply top and bottom by 3 to clear fractions.

$$\theta = 80.5^\circ \text{ (3 s.f.)}$$

4



G is the centre of mass if the framework

$$OG = \bar{x} = \frac{9(\sqrt{3} + 2)}{6 + \pi}$$

(see question 2 in Exercise 2E)

G is on the line of symmetry.

θ (see diagram) is the required angle.

$$\theta = 60^\circ - \hat{GAN}$$

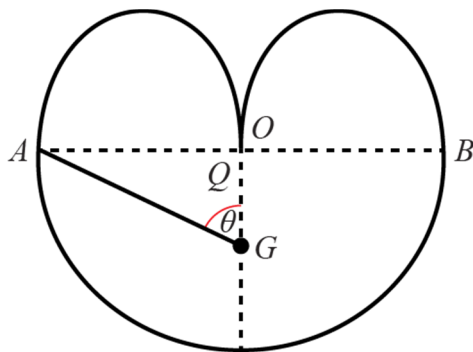
$$\begin{aligned} \tan \hat{GAN} &= \frac{GN}{AN} = \frac{6 \cos 30^\circ - \bar{x}}{6 \sin 30^\circ} \\ &= \frac{3\sqrt{3} - \bar{x}}{3} \\ &= \sqrt{3} - \frac{3(\sqrt{3} + 2)}{6 + \pi} \end{aligned}$$

$$\text{So, } \hat{GAN} = 26.898^\circ \dots$$

$$\text{So, } \theta = 33.1^\circ (3 \text{ s.f.})$$

AG will be vertical, when the framework hangs in equilibrium.

5



$$OG = \frac{3}{2\pi}, \text{ where } G \text{ is}$$

the centre of mass
so, AG will be vertical in equilibrium.

See question 4 in Exercise 2E.

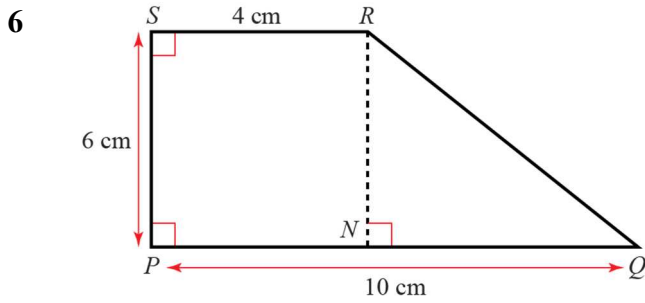
Since the angle with the horizontal will be 90° - angle with the vertical.

Required angle is $\hat{AGO} = \theta$

$$\begin{aligned} \tan \theta &= \frac{AO}{OG} \\ &= \frac{3}{\frac{3}{2\pi}} \\ &= 2\pi \end{aligned}$$

$$\theta = \tan^{-1}(2\pi)$$

$$= 81.0^\circ (3 \text{ s.f.})$$



Centre of mass of $\triangle RNQ$ has position vector

Taking PQ and PS as axes.

$$\frac{1}{3} \left\{ \begin{pmatrix} 4 \\ 0 \end{pmatrix} + \begin{pmatrix} 10 \\ 0 \end{pmatrix} + \begin{pmatrix} 4 \\ 6 \end{pmatrix} \right\} = \begin{pmatrix} 6 \\ 2 \end{pmatrix}$$

$$24 \begin{pmatrix} 2 \\ 3 \end{pmatrix} + 18 \begin{pmatrix} 6 \\ 2 \end{pmatrix} = 42 \begin{pmatrix} \bar{x} \\ \bar{y} \end{pmatrix}$$

Use $\sum m_i \mathbf{r}_i = \bar{\mathbf{r}} \sum m_i$

$$\begin{pmatrix} 48 \\ 72 \end{pmatrix} + \begin{pmatrix} 108 \\ 36 \end{pmatrix} = 42 \begin{pmatrix} \bar{x} \\ \bar{y} \end{pmatrix}$$

$$\begin{pmatrix} 156 \\ 108 \end{pmatrix} = 42 \begin{pmatrix} \bar{x} \\ \bar{y} \end{pmatrix}$$

Simplify.

$$\begin{pmatrix} \frac{26}{7} \\ \frac{18}{7} \end{pmatrix} = \begin{pmatrix} \bar{x} \\ \bar{y} \end{pmatrix}$$

So the centre of mass of the lamina is

a $\frac{26}{7}$ cm from PS and

b $\frac{18}{7}$ cm from PQ .

c We have

$$\tan \theta = \left(\frac{18}{7} \right) / \left(10 - \frac{26}{7} \right) = \frac{9}{22}$$

So $\theta = 22.2^\circ$

- 7 We choose coordinates so that the origin is at C and that BC lies on the x -axis, by considering the lamina as the union of a 2×6 rectangle and a 6×2 rectangle we see the centre of mass will satisfy

$$24 \begin{pmatrix} x \\ y \end{pmatrix} = 12 \begin{pmatrix} -1 \\ -3 \end{pmatrix} + 12 \begin{pmatrix} 3 \\ -5 \end{pmatrix}$$

So

$$\begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} 1 \\ -4 \end{pmatrix}$$

Now we attach a mass of $0.2M$ kg to F so the centre of mass of the whole system will satisfy

$$1.2M \begin{pmatrix} x \\ y \end{pmatrix} = M \begin{pmatrix} 1 \\ -4 \end{pmatrix} + 0.2M \begin{pmatrix} 6 \\ -6 \end{pmatrix}$$

So

$$1.2 \begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} 2.2 \\ -5.2 \end{pmatrix}$$

So

$$\begin{pmatrix} x \\ y \end{pmatrix} = \frac{1}{6} \begin{pmatrix} 11 \\ -26 \end{pmatrix}$$

Hence the angle θ satisfies

$$\tan \theta = \frac{26}{11}$$

So

$$\theta = 67.1^\circ$$

- 8 We choose coordinates so that the origin is at B and the x -axis is parallel to AC then the centre of mass of the lamina is

$$\begin{pmatrix} 0 \\ -\frac{32}{3\pi} \sin \frac{\pi}{4} \end{pmatrix}$$

And the coordinates of C are

$$\begin{pmatrix} 2\sqrt{2} \\ -2\sqrt{2} \end{pmatrix}$$

Hence the centre of mass of the system

Satisfies

$$1.5M \begin{pmatrix} x \\ y \end{pmatrix} = M \begin{pmatrix} 0 \\ -\frac{32}{3\pi} \sin \frac{\pi}{4} \end{pmatrix} + 0.5M \begin{pmatrix} 2\sqrt{2} \\ -2\sqrt{2} \end{pmatrix}$$

Hence

$$\begin{aligned} \begin{pmatrix} x \\ y \end{pmatrix} &= \frac{2}{3} \begin{pmatrix} \sqrt{2} \\ -\frac{16\sqrt{2}}{3\pi} - \sqrt{2} \end{pmatrix} = \frac{2\sqrt{2}}{3} \begin{pmatrix} 1 \\ -\frac{(16+3\pi)}{3\pi} \end{pmatrix} \\ &\approx \begin{pmatrix} 0.9428... \\ -2.5434... \end{pmatrix} \end{aligned}$$

Hence

$$\theta = 45^\circ + \tan^{-1} \left(\frac{0.9428}{2.5434} \right) = 45^\circ + 20.3^\circ = 65.3^\circ$$

- 9 a By considering the lamina as a union of two rectangles of size 2×4 and 4×2 we have the distance of the centre of mass to AC satisfies

$$16x = 8 \times 2 + 8 \times 5$$

So

$$16x = 56$$

$$x = \frac{7}{2}$$

- b Let T be the tension in the string at B then taking moments about A gives

$$6T = \frac{7}{2} \times 12g$$

So

$$T = 7g$$

Now let T be the tension in the string at A then taking moments about B gives

$$6T = \frac{5}{2} \times 12g$$

So

$$T = 5g$$

- c We will need to find the distance of the centre of mass from AB , by considering the rectangles that make up the lamina we have

$$16y = 8 \times 1 + 8 \times 2$$

So

$$y = 1.5$$

Hence the angle with the vertical is

$$\tan \theta = \frac{1.5}{2.5}$$

So $\theta = 31.0$

Hence the angle with the horizontal is

$$\varphi = 180^\circ - \theta = 149^\circ$$

10 We choose coordinates so that the origin is at the midpoint of PQ and the y -axis is parallel to PQ , then the coordinates of the centre of mass is

$$\left(\frac{4r}{3\pi}, 0\right)$$

So after attaching the mass at Q the centre of mass of the whole system satisfies

$$(1+k)\begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} \frac{4r}{3\pi} \\ 0 \end{pmatrix} + k\begin{pmatrix} 0 \\ -2r \end{pmatrix}$$

So

$$\begin{pmatrix} x \\ y \end{pmatrix} = \frac{1}{1+k}\begin{pmatrix} \frac{4r}{3\pi} \\ -2rk \end{pmatrix}$$

Now taking moments about P and the point where the right string meets the semicircle gives

$$rT_2 = \frac{4r}{3\pi(1+k)} \times (1+k)m$$

$$rT_1 = \left(r - \frac{4r}{3\pi(1+k)}\right) \times (1+k)m$$

Now using $T_1 = 5T_2$ gives

$$T_2 = \frac{4m}{3\pi}$$

$$5T_2 = \left(1 - \frac{4}{3\pi(1+k)}\right) \times (1+k)m$$

So

$$\frac{20}{3\pi} = \left(1 - \frac{4}{3\pi(1+k)}\right) \times (1+k)$$

So

$$k = \frac{8}{\pi} - 1$$

- 11 a** We take A to be the origin, then the lamina can be seen as a rectangle with a circular section removed

$$\text{we have the centre of mass satisfies } (6a^2 - \frac{\pi}{4}a^2) \begin{pmatrix} x \\ y \end{pmatrix} = 6a^2 \begin{pmatrix} a \\ \frac{3a}{2} \end{pmatrix} - \frac{\pi}{4}a^2 \begin{pmatrix} 2a - \frac{2a \sin \frac{\pi}{4}}{3(\frac{\pi}{4})} \cos \frac{\pi}{4} \\ \frac{2a \sin \frac{\pi}{4}}{3(\frac{\pi}{4})} \sin \frac{\pi}{4} \end{pmatrix}$$

So

$$(6a^2 - \frac{\pi}{4}a^2) \begin{pmatrix} x \\ y \end{pmatrix} = 6a^2 \begin{pmatrix} a \\ \frac{3a}{2} \end{pmatrix} - \frac{\pi}{4}a^2 \begin{pmatrix} 2a - \frac{4a}{3\pi} \\ \frac{16a}{3\pi} \end{pmatrix}$$

So we have

$$(6 - \frac{\pi}{4}) \begin{pmatrix} x \\ y \end{pmatrix} = 6 \begin{pmatrix} a \\ \frac{3a}{2} \end{pmatrix} - \frac{\pi}{4} \begin{pmatrix} 2a - \frac{4a}{3\pi} \\ \frac{4a}{3\pi} \end{pmatrix}$$

So

$$\frac{24 - \pi}{4} \begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} 6a \\ 9a \end{pmatrix} - \begin{pmatrix} \frac{a\pi}{2} - \frac{a}{3} \\ \frac{a}{3} \end{pmatrix}$$

So

$$(24 - \pi) \begin{pmatrix} x \\ y \end{pmatrix} = \frac{1}{3} \begin{pmatrix} 76a - 6a\pi \\ 104a \end{pmatrix}$$

So

$$\begin{pmatrix} x \\ y \end{pmatrix} = \frac{1}{3(24 - \pi)} \begin{pmatrix} 76a - 6a\pi \\ 104a \end{pmatrix}$$

As required.

- b** Let T_1, T_2 be the tensions in left and right strings respectively, taking moments about B gives

$$2aT_2 = \frac{2a(38 - 3\pi)}{3(24 - \pi)} \times W$$

Hence

$$T_2 = \frac{(38 - 3\pi)W}{3(24 - \pi)}$$

Now taking moments about C gives

$$2aT_1 = \left(2a - \frac{2a(38 - 3\pi)}{3(24 - \pi)} \right) W$$

Hence

$$T_1 = \left(1 - \frac{(38 - 3\pi)}{3(24 - \pi)} \right) W = \frac{34W}{3(24 - \pi)}$$

- c** We have the angle θ satisfies

$$\tan \theta = \left(2a - \frac{2a(38 - 3\pi)}{3(24 - \pi)} \right) \left(3a - \frac{104a}{3(24 - \pi)} \right)^{-1}$$

Hence

$$\tan \theta = \left(2 - \frac{2(38 - 3\pi)}{3(24 - \pi)} \right) \left(3 - \frac{104}{3(24 - \pi)} \right)^{-1}$$

Which gives

$$\theta = 39.1^\circ$$

Challenge

We choose coordinates such that the origin is where the string meets the mobile and AH is parallel to the x -axis then by symmetry the centre of mass of the mobile lies on the line from O to the midpoint of AH and the y component satisfies

$$(50 + 50 + 300)y = 50 \times 5 + 50 \times 5$$

$$+ 300 \times -2.5$$

$$\text{So } 400y = -250$$

So

$$y = -\frac{5}{8}$$

Now the coordinates of A are $(-30, -5)$

So the centre of mass of the system satisfies

$$(M + m) \begin{pmatrix} x \\ y \end{pmatrix} = M \begin{pmatrix} 0 \\ -\frac{5}{8} \end{pmatrix} + m \begin{pmatrix} -30 \\ -5 \end{pmatrix}$$

So

$$\begin{pmatrix} x \\ y \end{pmatrix} = \frac{1}{8(M + m)} \begin{pmatrix} -240m \\ -5M - 40m \end{pmatrix}$$

Now we consider the angle that the line DE makes with the vertical this satisfies

$$\tan \theta = \frac{5M + 40m}{240m}$$

On the other hand, if G touches the ceiling then the triangle formed by O , G and the point where the cable meets the ceiling gives

$$\tan \theta = \frac{\sqrt{375}}{25}$$

So

$$\frac{5M + 40m}{240m} = \frac{\sqrt{375}}{25} = \frac{\sqrt{15}}{5}$$

Which gives

$$m = 0.0343M$$